The values given in the load table are based upon uniform, smooth operation, $\mathrm{K}_{\mathrm{HB}}=1.0$ and reliable grease lubrication. Since, in practice, the applications are very diverse, it is important to consider the given conditions by using appropriate factors $\mathrm{S}_{\mathrm{B}}$, $K_{A}, L_{K H B}$ and $f_{n}$ (see below).

## Formulas for Determining the Tangential Force

$a=\frac{v}{t_{b}}$
$\left[\mathrm{m} / \mathrm{s}^{2}\right]$
$F_{u}=\frac{m \cdot g+m \cdot a}{1000}$ (for lifting axie)
[kN]
$F_{u}=\frac{m \cdot g \cdot \mu+m \cdot a}{1000}$ (for driving axle) $[\mathrm{kN}]$
$F_{u \text { perm. }}=\frac{F_{u \text { Tab }}}{K_{A} \cdot S_{B} \cdot f_{n} \cdot L_{K H B}}$
[kN]

Formula dimensions see page ZD-3

## The Condition $F_{u}<F_{u \text { perm. }}$. Must be Fulfilled.

Load Factor $\mathrm{K}_{\mathrm{A}}$

| Drive | Type of load from the machines to be driven |  |  |
| :--- | :---: | :---: | :---: |
|  | Uniform | Medium Shocks | Heavy Shocks |
| Uniform |  | 1.25 | 1.75 |
| Light Shocks | 1.25 | 1.50 | 2.00 |
| Medium Shocks | 1.50 | 1.75 | 2.25 |

## Safety Coefficient $\mathbf{S}_{B}$

The safety coefficient should be allowed for according to experience ( $S_{B}=1.1$ to 1.4).

## Life-Time Factor $f_{n}$

considering of the peripheral speed of the pinion and lubrication.

| Lubrication |  | Continuous | Daily | Monthly |
| :--- | :---: | :--- | :--- | :---: |
| Peripheral <br> of Gearing |  |  |  |  |
| m/sec | $\mathrm{m} / \mathrm{min}$ |  |  |  |
| 0.5 | 30 | 0.85 | 0.95 |  |
| 1.0 | 60 | 0.95 | 1.10 | from |
| 1.5 | 90 | 1.00 | 1.20 | 3 |
| 2.0 | 120 | 1.05 | 1.30 | to |
| 3.0 | 180 | 1.10 | 1.50 | 10 |
| 5.0 | 300 | 1.25 | 1.90 |  |

## Linear Load Distribution Factor $\mathrm{L}_{\text {KHB }}$

The linear load distribution factor considers the contact stress, while it describes unintegrated load distribution over the tooth width $\left(\mathrm{L}_{K H B}=\sqrt{\mathrm{K}_{H B}}\right)$.
$L_{K H B}=1.1$ for counter bearing, e.g. Torque Supporter
$=1.2$ for preloaded bearings on the output shaft e.g. ATLANTA HT, HP and E servo-worm gear unit, BG bevel-gear unit
$=1.5$ for unpreloaded bearings on the output shaft e.g. ATLANTA B servo-worm gear unit

## Calculation Example

## Values Given

$\otimes$ Travelling Operation

| Mass to be Moved | $m=820 \mathrm{~kg}$ |
| :--- | :--- | :--- |
| Speed | $\mathrm{v}=2 \mathrm{~m} / \mathrm{s}$ |
| Acceleration Time | $\mathrm{t}_{\mathrm{b}}=1 \mathrm{~s}$ |
| Acceleration Due to Gravity $\mathrm{g}=9.81 \mathrm{~m} / \mathrm{s}^{2}$ |  |
| Coefficient of Friction | $\mu=0.1$ |
| Load Factor | $\mathrm{K}_{\mathrm{A}}=1.5$ |
| Life-Time Factor | $\mathrm{f}_{\mathrm{n}}=1.05$ (cont. Iubrication) |
| Safety Coefficient | $\mathrm{S}_{\mathrm{B}}=1.2$ |
| Linear Load | $\mathrm{L}_{\mathrm{KHB}}=1.5$ |

## Calculation Process

Results
$\begin{array}{lcc}a & =\frac{v}{t_{b}} & a=\frac{2}{1}\end{array}=2 \mathrm{~m} / \mathrm{s}^{2}$
Assumed feed force: rack C 45 , ind. hardened, straight tooth, module 3, pinion 16 MnCr 5 , case hardened, 20 teeth, page C-46 with Futab $=11.5 \mathrm{kN}$

$$
\begin{aligned}
F_{u \text { zul./per. }}= & \frac{F_{u \text { Tab }}}{K_{A} \cdot S_{B} \cdot f_{n} \cdot L_{K H \beta}} ; \\
& F_{u \text { zul./per. }}=\frac{11.5 \mathrm{kN}}{1.5 \cdot 1.2 \cdot 1.05 \cdot 1.5}=4.05 \mathrm{kN}
\end{aligned}
$$

## Condition

| F $_{\mathrm{u} \text { zul./per. }}>\mathrm{F}_{\mathrm{u}} ; 4.05 \mathrm{kN}>2.44 \mathrm{kN}$ | $=>$ fulfilled |  |
| :--- | ---: | ---: |
| Result: | Rack | 3430100 |
|  | Page C-64 |  |
|  | Pinion | 2435220 |
|  | Page C-40 |  |
| Case-Hardened |  |  |

## Your Calculation

## Values Given

$\otimes$ Travelling Operation

Coefficient of Friction $\quad \mu=$

Load Factor $\quad \mathrm{K}_{\mathrm{A}}=$ $\qquad$
Life-Time Factor
$f_{n}=$ $\qquad$
Safety Coefficient
$\mathrm{S}_{\mathrm{B}}=$ $\qquad$
Linear Load
Distribution Factor
$L_{K H \beta}=$ $\qquad$

## Calculation Process

Results

$$
\begin{array}{ll}
a=\frac{v}{t_{b}} & a=\square=\square \mathrm{m} / \mathrm{s}^{2} \\
F_{u}=\frac{m \cdot g \cdot \mu+m \cdot a}{1000} ; & F_{u}=\frac{1000}{}=\square \mathrm{kN}
\end{array}
$$

Permissible Feed Force $F_{u}$ Tab
$F_{u \text { zul./per. }}=\frac{F_{u \text { Tab }}}{K_{A} \cdot S_{B} \cdot f_{n} \cdot L_{K H \beta}}$;

$$
F_{u \text { zul. /per. }}=
$$

$\qquad$ $=$ $\qquad$ kN

## Condition

$F_{\mathrm{u} \text { zul./per. }}>\mathrm{F}_{\mathrm{u}} ; \quad \ldots \quad \mathrm{kN}>\ldots \quad \mathrm{kN} \quad=>$ fulfilled

## Calculation Example

## Values Given

O Lifting Operation

| Mass to be Moved | $m=300 \mathrm{~kg}$ |
| :--- | :--- |
| Speed | $\mathrm{v}=1.08 \mathrm{~m} / \mathrm{s}$ |
| Acceleration Time | $\mathrm{t}_{\mathrm{b}}=0.27 \mathrm{~s}$ |
| Acceleration Due to Gravity g | $=9.81 \mathrm{~m} / \mathrm{s}^{2}$ |
| Load Factor | $\mathrm{K}_{\mathrm{A}}=1.2$ |
| Life-Time Factor | $\mathrm{f}_{\mathrm{n}}=1.1$ (Cont. Lubrication) |
| Safety Coefficient | $\mathrm{S}_{\mathrm{B}}=1.2$ |
| Linear Load | $\mathrm{L}_{\mathrm{KH} \beta}=1.2$ |
| Distribution Factor |  |

## Calculation Process

## Results

| $a=\frac{v}{t_{b}}$ | $a=\frac{1.08}{0.27}$ | $=4 \mathrm{~m} / \mathrm{s}^{2}$ |
| ---: | :--- | ---: |
| $F_{u}=\frac{m \cdot g+m \cdot a}{1000}$ | $u=\frac{300 \cdot 9.81+300 \cdot 4}{1000}$ | $=4.1 \mathrm{kN}$ |

Assumed feed force: rack C45, ind. hardened, helical, module 2, pinion 16 MnCr 5 , case hardened, 20 teeth, page $\mathrm{C}-45$ with $\mathrm{F}_{\text {utab }}=12 \mathrm{kN}$
$F_{u \text { zul./per. }}=\frac{F_{u \text { Tab }}}{K_{A} \cdot S_{B} \cdot f_{n} \cdot L_{K H \beta}} ; F_{u \text { zul./per. }}=\frac{11.5 \mathrm{kN}}{1.2 \cdot 1.2 \cdot 1.1 \cdot 1.2}$

$$
=5.9 \mathrm{kN}
$$

## Condition

| $\mathrm{F}_{\mathrm{uzul} \text {./per. }}>\mathrm{F}_{\mathrm{u}} ; 6.0 \mathrm{kN}>4.1 \mathrm{kN}$ | $=>$ fulfilled |  |
| :--- | ---: | ---: |
| Result: | Rack | 2920105 | Page C-16

## Your Calculation

## Values Given

O Lifting Operation
Mass to be Moved $m=$

Speed
v = $\qquad$ $\mathrm{m} / \mathrm{s}$

Acceleration Time
$t_{b}=$ $\qquad$ s

Acceleration Due to Gravity g = $\qquad$ $\mathrm{m} / \mathrm{s}^{2}$

Load Factor
Life-Time Factor
Safety Coefficient
Linear Load
Distribution Factor
$\mathrm{K}_{\mathrm{A}}=$ $\qquad$
$\mathrm{f}_{\mathrm{n}}=$ $\qquad$
$\mathrm{S}_{\mathrm{B}}=$ $\qquad$
$\mathrm{L}_{\mathrm{KH} \beta}=$ $\qquad$

## Calculation Process

Results

| $a$ | $=\frac{v}{t_{b}}$ | $a=\ldots \quad \mathrm{m} / \mathrm{s}^{2}$ |  |
| ---: | :--- | ---: | :--- |
| $F_{u}$ | $=\frac{m \cdot g+m \cdot a}{1000}$ | $F_{u \text { erf./req. }}=\ldots$ |  |

Permissible Feed Force $\mathrm{F}_{\mathrm{u} \text { tab }}$
$F_{\mathrm{u} \text { zul./per. }}=\frac{F_{u \text { Tab }}}{K_{A} \cdot S_{B} \cdot f_{n} \cdot L_{K H \beta}} ; F_{u \text { zul./per. }}=$ $\qquad$ $=$ $\qquad$ kN

## Condition

$\mathrm{F}_{\mathrm{u} \text { zul./per. }}>\mathrm{F}_{\mathrm{u}} ; \quad \mathrm{kN}>\quad \mathrm{kN} \quad=>$ fulfilled

